#### Controller Design and Tuning

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- Our example (closed loop): velocity control in rail car

Moving Car (the physical "plant") Model

$$F_{res} = F_{traction} + F_{drag}$$

$$F_{drag} = -\frac{1}{2} \cdot p \cdot v^2 \cdot C_D \cdot A$$

$$F_{res} = M \cdot a = M \cdot \frac{dv}{dt} \qquad \xleftarrow{F_{drag}} \qquad \xleftarrow{F_{traction}} \\ \xleftarrow{F_{res}} = \frac{1}{2} \cdot p \cdot v^2 \cdot C_D \cdot A$$

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$$\frac{dv}{dt} = \frac{1}{M} \left( F_{traction} - \frac{1}{2} \cdot p \cdot v^{2} \cdot C_{D} \cdot A \right)$$
$$v(0) = 0$$

A Proportional-Integral-Derivative (PID) controller takes as input the error (deviation of the measured/sensed value from the ideal or "setpoint" value)  $v_i - v$  and produces an output to be sent to the plant via the actuator.

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A PID controller produces a control output:

$$K_p \cdot (v_i - v) + K_i \cdot \int (v_i - v) dt + K_d \cdot \frac{d(v_i - v)}{dt}$$

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## Closed-Loop PID Controller for Velocity Control





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- Passengers should not fall (*i.e.*, accelerate too much).
- Other requirements such as minimizing total energy consumption could be added.

## Abstracting the Passenger: Mass-Spring-Damper System

$$\begin{cases} F_{ext} = -f \\ F_{spring} = -k(-x) \\ F_{damper} = -c(-v) \\ M \cdot a = F_{ext} + F_{spring} + F_{damper} \\ \frac{dv}{dt} = a \\ \frac{dx}{dt} = v \\ \end{cases}$$

$$\begin{cases} \frac{dv}{dt} &= \frac{1}{M}(-f + k \cdot x + c \cdot v) \\ \frac{dx}{dt} &= v \end{cases}$$

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#### Abstracting Train-and-Passenger ("Plant" model)



#### Complete System Overview



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#### Some Results - Train Velocity



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#### Some Results - Passenger Displacement and Acceleration



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